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Contact between elastic bodies. III. Dual finite element analysis

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CONTACT BETWEEN ELASTIC BODIES — III. DUAL FINITE  
ELEMENT ANALYSIS

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In Part I and II of this paper, the primary variational formulation of the contact problem — i.e., in terms of displacements — has been studied. In the present part, we use a dual variational formulation of the same problem, — i.e., in terms of stresses. For simplicity, we restrict ourselves to the problems with a bounded contact zone and with polygonal boundaries.

The dual formulation — i.e. principle of minimum complementary energy is deduced in Section 1 via the theory of a saddle-point. In contradistinction to the primary principle — i.e. minimum of potential energy — we obtain a uniquely solvable formulation even in case that the primary formulation has an infinity of solutions.

In section 2 we define approximations to the dual variational problem on the basis of self-equilibrated triangular block-elements, which have been proposed by Watwood and Hartz [3] and studied in [4] and [5]. An a priori  $L_2$ -error estimate is proved under a hypothesis that the solution is sufficiently regular. An algorithm for the approximations is presented in Section 3.

1. DUAL VARIATIONAL FORMULATION OF THE CONTACT PROBLEM  
WITH A BOUNDED CONTACT ZONE AND ZERO FRICTION

Starting with the primary variational formulation — the principle of minimum potential energy — we derive a dual formulation — the principle of minimum complementary energy — via theory of a saddle-point. To this end, we recall first the concept of the saddle-point.

Let  $\mathcal{A}, \mathcal{B}$  be two non-empty sets,  $\mathcal{H}$  a real functional, defined on  $\mathcal{A} \times \mathcal{B}$ . The pair  $\{u, \lambda\}$  will be called a *saddle point* of  $\mathcal{H}$  on  $\mathcal{A} \times \mathcal{B}$ , if

$$(1.1) \quad \mathcal{H}(u, \mu) \leq \mathcal{H}(u, \lambda) \leq \mathcal{H}(v, \lambda) \quad \forall v \in \mathcal{A}, \quad \forall \mu \in \mathcal{B}.$$

**Lemma 1.1.** Let  $\{u, \lambda\}$  be a saddle point of  $\mathcal{H}$  on  $\mathcal{A} \times \mathcal{B}$ . Then

$$\mathcal{H}(u, \lambda) = \inf_{v \in \mathcal{A}} \sup_{\mu \in \mathcal{B}} \mathcal{H}(v, \mu) = \sup_{\mu \in \mathcal{B}} \inf_{v \in \mathcal{A}} \mathcal{H}(v, \mu).$$

For the proof – see e.g. [2] – chpt. 5.

Let us recall the primary variational formulation of the contact problem (cf. [1] – Section 1). We introduced the following notation

$$\begin{aligned} W &= \{ \mathbf{u} \mid \mathbf{u} = (\mathbf{u}', \mathbf{u}'') \in [H^1(\Omega')]^2 \times [H^1(\Omega'')]^2 \}, \\ V &= \{ \mathbf{u} \in W \mid u'_n = 0 \text{ on } \Gamma_u, u''_n = 0 \text{ on } \Gamma_0 \}, \\ K &= \{ \mathbf{v} \in V \mid v'_n + v''_n \leq 0 \text{ on } \Gamma_K \}, \\ A(\mathbf{u}, \mathbf{v}) &= \int_{\Omega' \cup \Omega''} c_{ijkl} e_{ij}(\mathbf{u}) e_{km}(\mathbf{v}) \, d\mathbf{x}, \\ L(\mathbf{v}) &= \int_{\Omega' \cup \Omega''} F_i v_i \, d\mathbf{x} + \int_{\Gamma_{\tau'} \cup \Gamma_{\tau''}} P_i v_i \, ds, \\ \mathcal{L}(\mathbf{v}) &= \frac{1}{2} A(\mathbf{v}, \mathbf{v}) - L(\mathbf{v}). \end{aligned}$$

The primary problem is to find  $\mathbf{u} \in K$  such that

$$(1.2) \quad \mathcal{L}(\mathbf{u}) \leq \mathcal{L}(\mathbf{v}) \quad \forall \mathbf{v} \in K.$$

Let us introduce a set  $S$  of new parameters as follows:

$$\begin{aligned} S &= \{ \mathcal{N} = (\mathcal{N}_{ij}); i, j = 1, 2, \mathcal{N}_{ij} \in L_2(\Omega), \mathcal{N}_{12} = \mathcal{N}_{21} \}, \\ \Omega &= \Omega' \cup \Omega''. \end{aligned}$$

Setting

$$(1.3) \quad \mathcal{N}_{ij} = e_{ij}(\mathbf{v}), \quad i, j = 1, 2,$$

we may write

$$\mathcal{L}(\mathbf{v}) = \frac{1}{2} \int_{\Omega} c_{ijkl} \mathcal{N}_{ij} \mathcal{N}_{kl} \, d\mathbf{x} - L(\mathbf{v}) = \mathcal{L}_1(\mathcal{N}, \mathbf{v})$$

and the primary problem (1.2) is equivalent to the minimization of  $\mathcal{L}_1(\mathcal{N}, \mathbf{v})$  with the constraints (1.3). The latter problem can be solved by means of Lagrange multipliers  $\lambda$  and the following Lagrangian  $\mathcal{H}$ :

$$\mathcal{H}([\mathcal{N}, \mathbf{v}], \lambda) = \mathcal{L}_1(\mathcal{N}, \mathbf{v}) + \int_{\Omega} \lambda_{ij} (e_{ij}(\mathbf{v}) - \mathcal{N}_{ij}) \, d\mathbf{x}.$$

It is readily seen that

$$\sup_{\lambda \in S} \int_{\Omega} \lambda_{ij} (e_{ij}(\mathbf{v}) - \mathcal{N}_{ij}) \, d\mathbf{x} = \begin{cases} 0 & \text{if } \mathcal{N}_{ij} = e_{ij}(\mathbf{v}) \quad \forall i, j \\ +\infty & \text{if not.} \end{cases}$$

Consequently, any solution  $\mathbf{u}$  of the primary problem (1.2) satisfies the relation

$$(1.4) \quad \mathcal{L}(\mathbf{u}) = \inf_{\mathbf{v} \in K} \mathcal{L}(\mathbf{v}) = \inf_{\substack{\mathbf{v} \in K \\ \mathcal{N} \in S}} \sup_{\lambda \in S} \mathcal{H}([\mathcal{N}, \mathbf{v}], \lambda).$$

The following variational problem

$$(1.5) \quad \sup_{\lambda \in S} \inf_{[\mathcal{N}, \mathbf{v}] \in S \times K} \mathcal{H}([\mathcal{N}, \mathbf{v}], \lambda)$$

is called *dual* to the (primary) problem (1.4). It is desirable to investigate the relation between the values of (1.4) and (1.5). To this end we apply lemma 1.1. First we prove

**Lemma 1.2.** *Let  $\{[\mathcal{N}^*, \mathbf{v}^*], \lambda^*\}$  be a saddle point of  $\mathcal{H}$  on  $\mathcal{W} \times S$ , where  $\mathcal{W} = S \times K$ . Then a solution  $\mathbf{u}$  of the primary problem exists and*

$$\mathcal{N}^* = \mathbf{e}(\mathbf{u}), \quad \mathbf{v}^* = \mathbf{u}, \quad \lambda^* = \tau(\mathbf{u}),$$

where  $\mathbf{e}(\mathbf{u})$  and  $\tau(\mathbf{u})$  are the corresponding strain and stress tensor, respectively.

*Proof.* From (1.1) we deduce:

$$(1.6) \quad \delta_\lambda \mathcal{H}([\mathcal{N}^*, \mathbf{v}^*], \lambda^*) = 0 \Leftrightarrow \mathcal{N}_{ij}^* = e_{ij}(\mathbf{v}^*),$$

$$(1.7) \quad \delta_{\mathcal{N}} \mathcal{H}([\mathcal{N}^*, \mathbf{v}^*], \lambda^*) = 0 \Leftrightarrow \lambda_{ij}^* = c_{ijkl} \mathcal{N}_{kl}^*,$$

$$(1.8) \quad \delta_{\mathbf{v}} \mathcal{H}([\mathcal{N}^*, \mathbf{v}^*], \lambda^*) (\mathbf{v} - \mathbf{v}^*) \geq 0 \quad \forall \mathbf{v} \in K$$

(where e.g.  $\delta_\lambda \mathcal{H}$  denotes the partial Gâteaux derivative of  $\mathcal{H}$  with respect to  $\lambda$ ). The inequality (1.8) together with (1.6) and (1.7) implies

$$A(\mathbf{v}^*, \mathbf{v} - \mathbf{v}^*) \geq L(\mathbf{v} - \mathbf{v}^*) \quad \forall \mathbf{v} \in K.$$

Consequently,  $\mathbf{v}^*$  is a solution of the problem (1.2).

**Lemma 1.3.** *Let  $\mathbf{u}$  be a solution of the primary problem. Then  $\{[\mathbf{e}(\mathbf{u}), \mathbf{u}], \tau(\mathbf{u})\}$  is a saddle point of  $\mathcal{H}$  on  $\mathcal{W} \times S$ .*

*Proof.* We have to verify that

$$\mathcal{H}([\mathbf{e}(\mathbf{u}), \mathbf{u}], \mu) \leq \mathcal{H}([\mathbf{e}(\mathbf{u}), \mathbf{u}], \tau(\mathbf{u})) \leq \mathcal{H}([\mathcal{N}, \mathbf{v}], \tau(\mathbf{u}))$$

holds for any  $\mu \in S$  and  $[\mathcal{N}, \mathbf{v}] \in \mathcal{W}$ .

The left-hand inequality turns to an equality due to the definition of  $\mathcal{H}$ . The right-hand inequality can be written in the following form

$$\begin{aligned} & \frac{1}{2} \int_{\Omega} c_{ijkl} e_{ij}(\mathbf{u}) e_{kl}(\mathbf{u}) \, d\mathbf{x} - L(\mathbf{u}) \leq \\ & \leq \frac{1}{2} \int_{\Omega} c_{ijkl} \mathcal{N}_{ij} \mathcal{N}_{kl} \, d\mathbf{x} - L(\mathbf{v}) + \int_{\Omega} c_{ijkl} e_{kl}(\mathbf{u}) (e_{ij}(\mathbf{v}) - \mathcal{N}_{ij}) \, d\mathbf{x}, \end{aligned}$$

which is equivalent to

$$\begin{aligned} & \frac{1}{2} \int_{\Omega} c_{ijkl} (\mathcal{N}_{ij} - e_{ij}(\mathbf{u})) (\mathcal{N}_{kl} - e_{kl}(\mathbf{u})) \, d\mathbf{x} + \\ & + \int_{\Omega} c_{ijkl} e_{kl}(\mathbf{u}) e_{ij}(\mathbf{v} - \mathbf{u}) \, d\mathbf{x} \geq L(\mathbf{v} - \mathbf{u}) \quad \forall \mathbf{v} \in K, \quad \forall \mathcal{N} \in S. \end{aligned}$$

The latter inequality, however, is an easy consequence of the ellipticity of the coefficients  $c_{ijkl}$  (cf. (1.4) in [1] – 1) and the definition of  $\mathbf{u}$ .

**Lemma 1.4.** *Let a solution  $\mathbf{u}$  of the primary problem (1.2) exist. Then*

$$(1.9) \quad \mathcal{L}(\mathbf{u}) = \inf_{\substack{\mathcal{N} \in S \\ \mathbf{v} \in K}} \sup_{\lambda \in S} \mathcal{H}([\mathcal{N}, \mathbf{v}], \lambda) = \sup_{\lambda \in S} \inf_{\substack{\mathcal{N} \in S \\ \mathbf{v} \in K}} \mathcal{H}([\mathcal{N}, \mathbf{v}], \lambda).$$

*Proof.* Using (1.4), Lemma 1.3 and Lemma 1.1, the assertion follows.

Next let us simplify the dual problem (1.5) by an elimination of the variables  $[\mathcal{N}, \mathbf{v}]$ . Consider the „inner” problem

$$\inf_{[\mathcal{N}, \mathbf{v}] \in \mathcal{W}} \mathcal{H}([\mathcal{N}, \mathbf{v}], \lambda),$$

where  $\lambda \in S$  is a fixed element. Obviously, we have

$$(1.10) \quad \inf_{\mathcal{W}} \mathcal{H} = \inf_{\mathcal{N} \in S} \mathcal{H}_1(\mathcal{N}, \lambda) + \inf_{\mathbf{v} \in K} \mathcal{H}_2(\mathbf{v}, \lambda)$$

where

$$(1.11) \quad \begin{aligned} \mathcal{H}_1(\mathcal{N}, \lambda) &= \frac{1}{2} \int_{\Omega} c_{ijkl} \mathcal{N}_{ij} \mathcal{N}_{kl} \, d\mathbf{x} - \int_{\Omega} \lambda_{ij} \mathcal{N}_{ij} \, d\mathbf{x}, \\ \mathcal{H}_2(\mathbf{v}, \lambda) &= \int_{\Omega} \lambda_{ij} e_{ij}(\mathbf{v}) \, d\mathbf{x} - L(\mathbf{v}). \end{aligned}$$

We can find easily that

$$(1.12) \quad \inf_{\mathcal{N} \in S} \mathcal{H}_1(\mathcal{N}, \lambda) = \mathcal{H}_1(\mathcal{N}^*, \lambda) = -\frac{1}{2} \int_{\Omega} a_{ijkl} \lambda_{ij} \lambda_{kl} \, d\mathbf{x},$$

where  $a_{ijkl}$  are coefficients of the inverse generalized Hooke’s law and  $\mathcal{N}_{ij}^* = a_{ijkl} \lambda_{kl}$ .

Let there exist a  $\mathbf{v}_0 \in K$  such that  $\mathcal{H}_2(\mathbf{v}_0, \lambda) < 0$ . Since  $K$  is a convex cone,  $t\mathbf{v}_0 \in K$  and

$$\mathcal{H}_2(t\mathbf{v}_0, \lambda) \rightarrow -\infty \quad \text{for } t \rightarrow +\infty.$$

If  $\mathcal{H}_2(\mathbf{v}, \lambda) \geq 0$  for all  $\mathbf{v} \in K$ , then

$$\inf_{\mathbf{v} \in K} \mathcal{H}_2(\mathbf{v}, \lambda) = \mathcal{H}_2(0, \lambda) = 0.$$

Denote

$$(1.13) \quad K_{F,P}^+ = \{\lambda \in S \mid \mathcal{H}_2(\mathbf{v}, \lambda) \geq 0 \quad \forall \mathbf{v} \in K\}.$$

We thus obtained that

$$(1.14) \quad \inf_{\mathbf{v} \in K} \mathcal{H}_2(\mathbf{v}, \lambda) = \begin{cases} 0 & \text{if } \lambda \in K_{F,P}^+, \\ -\infty & \text{if } \lambda \notin K_{F,P}^+. \end{cases}$$

**Lemma 1.5.** *Let a solution  $\mathbf{u}$  of the primary problem exist. Then  $K_{F,P}^+$  is a non-empty, closed and convex subset of  $S$ .*

*Proof.* We show that  $\tau(\mathbf{u})$  belongs to  $K_{F,P}^+$ . In fact,

$$\int_{\Omega} \tau_{ij}(\mathbf{u}) e_{ij}(\mathbf{v} - \mathbf{u}) \, d\mathbf{x} \geq L(\mathbf{v} - \mathbf{u}) \quad \forall \mathbf{v} \in K$$

holds and substituting  $\mathbf{v} = \mathbf{u} + \mathbf{w}$ , where  $\mathbf{w}$  is any element of  $K$ , we obtain

$$\mathcal{H}_2(\mathbf{w}, \tau(\mathbf{u})) = \int_{\Omega} \tau_{ij}(\mathbf{u}) e_{ij}(\mathbf{w}) - L(\mathbf{w}) \geq 0 \quad \forall \mathbf{w} \in K.$$

The convexity and closedness of  $K_{F,P}^+$  is easy to see.

Q.E.D.

From (1.10), (1.12) and (1.14) it follows that

$$\inf_{[\mathcal{N}, \mathbf{v}] \in \mathcal{W}} \mathcal{H}([\mathcal{N}, \mathbf{v}], \lambda) = \begin{cases} -\mathcal{S}(\lambda) & \text{if } \lambda \in K_{F,P}^+, \\ -\infty & \text{if } \lambda \notin K_{F,P}^+ \end{cases}$$

where

$$\mathcal{S}(\lambda) = \frac{1}{2} \int_{\Omega} a_{ijkl} \lambda_{ij} \lambda_{kl} \, d\mathbf{x}.$$

Consequently, we have

$$\sup_{\lambda \in S} \inf_{[\mathcal{N}, \mathbf{v}] \in \mathcal{W}} \mathcal{H}([\mathcal{N}, \mathbf{v}], \lambda) = \sup_{\lambda \in K_{F,P}^+} (-\mathcal{S}(\lambda)) = - \inf_{\lambda \in K_{F,P}^+} \mathcal{S}(\lambda).$$

Moreover, if a saddle point of  $\mathcal{H}$  on  $\mathcal{W} \times S$  exists, then

$$(1.15) \quad - \inf_{\lambda \in K_{F,P}^+} \mathcal{S}(\lambda) = -\mathcal{S}(\tau(\mathbf{u})) = \mathcal{L}(\mathbf{u}),$$

where  $\mathbf{u}$  is a solution to the primary problem. In fact, the assertion follows from Lemmas 1.2, 1.1 and 1.4.

On the other hand, Lemma 1.3 guarantees the existence of a saddle point, provided a solution of the primary problem exists.

In [1] – I, Section 2, we studied the existence and uniqueness of weak solutions of the primary problem  $\mathcal{P}_1$ , i.e., (1.2). Under the assumptions of Theorem 2.2 there, i.e. if

$$(1.16) \quad \begin{aligned} L(y) &\leq 0 \quad \forall \mathbf{y} \in K \cap \mathcal{R}, \\ L(y) &< 0 \quad \forall \mathbf{y} \in K \cap \mathcal{R} \div \mathcal{R}^*, \end{aligned}$$

a solution  $\mathbf{u}$  of the primary problem exists. Since the difference of any two solutions belongs to the subspace  $\mathcal{R}$  of rigid displacements, both  $e(\mathbf{u})$  and  $\tau(\mathbf{u})$  are determined uniquely. Consequently, if (1.16) holds, the saddle point of  $\mathcal{H}$  exists, its last component is uniquely determined and (1.15) is true. In other words, the *dual problem* to find  $\lambda \in K_{F,p}^+$  such that

$$(1.17) \quad \mathcal{S}(\lambda) \leq \mathcal{S}(\mu) \quad \forall \mu \in K_{F,p}^+,$$

has a *unique* solution  $\lambda = \tau(\mathbf{u})$ , where  $\mathbf{u}$  is any solution of the primary problem.

Thus we obtained a *uniquely* solvable formulation of a general class of contact problems in contradistinction to the primary variational formulation (see Section 2 of [1] – I, where only the cases of one-dimensional spaces of rigid virtual displacements have been considered).

**Remark 1.1.** The existence and uniqueness of the solution of (1.17) can be proved directly, using Lemma 1.5 and the strict convexity of the functional  $\mathcal{S}$ .

**Remark 1.2.** Let us emphasize that the dual problem is uniquely solvable, whenever the primary problem possesses a solution. The existence (and uniqueness) of the solution for the dual problem, however, follows directly, if the set  $K_{F,p}^+$  is non-empty. Thus a possibility arises that the dual problem may have a solution even in some cases when the primary problem has not. We shall not investigate the latter question, but remark only that the set  $K_{F,p}^+$  is non-empty only if the condition  $(1.16)_1$  is satisfied. Consequently,  $(1.16)_1$  is a necessary condition for the existence of a solution for both the primary and the dual problem.

Interpretation of the set  $K_{F,p}^+$

For the purposes of an approximation, it is useful to study the set  $K_{F,p}^+$  of admissible stress fields more closely.

**Lemma 1.6.**  $1^\circ$  Let  $\lambda \in K_{F,p}^+$  be sufficiently smooth. Then  $\lambda$  satisfies the following conditions:

$$(1.18) \quad \frac{\partial \lambda_{ij}}{\partial x_j} + F_i = 0 \quad \text{in } \Omega = \Omega' \cup \Omega'', \quad i = 1, 2, ,$$

$$(1.19) \quad \lambda_{ij} n_j = P_i \quad \text{on } \Gamma_\tau = \Gamma'_\tau \cup \Gamma''_\tau, \quad i = 1, 2, ,$$

$$(1.20) \quad T_i(\lambda) = 0 \quad \text{on } \Gamma_0, ,$$

$$(1.21) \quad T_i(\lambda') = T_i(\lambda'') = 0 \quad \text{on } \Gamma_K, ,$$

$$(1.22) \quad T_n(\lambda') = T_n(\lambda'') \leq 0 \quad \text{on } \Gamma_K. ,$$

$2^\circ$  Conversely, let  $\lambda \in S$  be sufficiently smooth and let it satisfy (1.18)–(1.22). Then  $\lambda \in K_{F,p}^+$ .

Proof. 1° Integrating by parts we obtain for any  $\mathbf{v} \in K$

$$\int_{\Omega} \lambda_{ij} e_{ij}(\mathbf{v}) \, d\mathbf{x} = - \int_{\Omega} v_i \frac{\partial \lambda_{ij}}{\partial x_j} + \int_{\partial\Omega' \cup \partial\Omega''} (T_n(\lambda) v_n + T_i(\lambda) v_i) \, ds \geq \int_{\Omega} F_i v_i \, d\mathbf{x} + \int_{\Gamma_{\tau}} P_i v_i \, ds.$$

Inserting  $v_i^M = \pm \varphi_i \in C_0^{\infty}(\Omega^M)$ ,  $M = ', ''$ , we are led to (1.18). Hence we obtain

$$\int_{\partial\Omega' \cup \partial\Omega''} T_i(\lambda) v_i \, ds \geq \int_{\Gamma_{\tau}} P_i v_i \, ds \quad \forall \mathbf{v} \in K.$$

Choosing  $v_i = \pm \psi_i$  such that the traces of  $\psi$  have their support in  $\Gamma_{\tau}$ , we obtain (1.19).

Let us choose  $\mathbf{v}' = 0$ ,  $\mathbf{v}''$  with  $v_n'' = 0$ ,  $v_i'' = \pm \psi$  on  $\Gamma_0$ , where the support of  $\psi$  is in  $\Gamma_0$ . Consequently, (1.20) follows.

It remains to analyze the inequality

$$(1.23) \quad \int_{\Gamma_K} (T_n(\lambda') v_n' + T_i(\lambda') v_i' + T_n(\lambda'') v_n'' + T_i(\lambda'') v_i'') \, ds \geq 0 \quad \forall \mathbf{v} \in K.$$

Choosing  $\mathbf{v} \in V$  such that  $v_i' = v_i'' = 0$  and  $v_n' = -v_n'' = \pm \varphi$  on  $\Gamma_K$ ,  $\varphi \in C_0^{\infty}(\Gamma_K)$ , we obtain

$$\int_{\Gamma_K} (T_n(\lambda') - T_n(\lambda'')) \varphi \, ds = 0.$$

Hence

$$T_n(\lambda') = T_n(\lambda'') \quad \text{on } \Gamma_K.$$

Let us choose  $\mathbf{v} \in V$  such that  $v_n' = v_n'' = 0$ ,  $v_i'' = 0$ ,  $v_i' = \pm \varphi$  on  $\Gamma_K$ . Then (1.23) yields  $T_i(\lambda') = 0$  on  $\Gamma_K$ . The condition  $T_i(\lambda'') = 0$  can be derived in a similar way.

It remains

$$\int_{\Gamma_K} T_n(\lambda) (v_n' + v_n'') \, ds \geq 0 \quad \forall \mathbf{v} \in K$$

and  $T_n(\lambda) \leq 0$  follows, using the condition  $v_n' + v_n'' \leq 0$  on  $\Gamma_K$ .

2° Let us multiply (1.18) by a function  $v_i$ , where  $\mathbf{v} \in K$ , and integrate over  $\Omega$  by parts. We obtain

$$\begin{aligned} 0 &= - \int_{\Omega} \lambda_{ij} \frac{\partial v_i}{\partial x_j} \, d\mathbf{x} + \int_{\Omega} F_i v_i \, d\mathbf{x} + \int_{\partial\Omega' \cup \partial\Omega''} \lambda_{ij} n_j v_i \, ds = \\ &= - \int_{\Omega} \lambda_{ij} e_{ij}(\mathbf{v}) \, d\mathbf{x} + L(\mathbf{v}) + \int_{\Gamma_K} T_n(\lambda) (v_n' + v_n'') \, ds. \end{aligned}$$

From the definition of  $K$  and (1.22) we conclude that the last integral is non-negative and consequently,  $\lambda \in K_{F,p}^+$ .



2. APPROXIMATIONS OF THE DUAL PROBLEM WITH A BOUNDED CONTACT ZONE

As usually, finite-dimensional approximations of the set  $K_{F,P}^+$  of admissible stress-functions are needed. To this end, it is suitable first to find a „particular” solution  $\bar{\lambda}$  of the non-homogeneous conditions (1.18), (1.19) and then to write

$$\lambda = \bar{\lambda} + \tau,$$

where  $\tau$  are self-equilibrated stress fields.

Since the system of forces  $F_i, P_i$  acting on  $\Omega''$  has a non-zero resultant, it is necessary to introduce reaction forces (normal vector loading) on  $\Gamma_K$ , with respect to both  $\Omega''$  and  $\Omega'$ .

**Lemma 2.1.** *If  $\bar{\lambda} \in S$  satisfies (1.18), (1.19), then*

$$(2.1) \quad \int_{\Gamma_K} T_i(\bar{\lambda}'') \, ds = - \left( \int_{\Omega''} F_i \, d\mathbf{x} + \int_{\Gamma_{\tau''}} P_i \, ds + \int_{\Gamma_0} T_i(\bar{\lambda}'') \, ds \right), \quad i = 1, 2.$$

*Proof.* Using (1.18) and (1.19), we may write

$$- \int_{\Omega''} F_i \, d\mathbf{x} = \int_{\Omega''} \frac{\partial \bar{\lambda}_{ij}''}{\partial x_j} \, d\mathbf{x} = \int_{\partial\Omega''} \bar{\lambda}_{ij}'' n_j'' \, ds = \int_{\Gamma_{\tau''}} P_i \, ds + \int_{\Gamma_0} T_i(\bar{\lambda}'') \, ds + \int_{\Gamma_K} T_i(\bar{\lambda}'') \, ds,$$

and (2.1) follows. Q.E.D.

Observing Lemma 2.1, we can choose a simplest distribution of the reaction forces  $T_n(\bar{\lambda})$  on  $\Gamma_K$ .

*Example 1.* Let  $\Gamma_0$  consist of straight segments parallel with  $x_1$ -axis and  $\Gamma_K$  be a straight segment such that  $n_1'' > 0$  on  $\Gamma_K$ . We can choose  $\bar{\lambda} \in K_{F,P}^+$  such that

$$(2.2) \quad T_n(\bar{\lambda}') = T_n(\bar{\lambda}'') = g \quad \text{on} \quad \Gamma_K,$$

where

$$(2.3) \quad g = - \left( \int_{\Omega''} F_1 \, dx + \int_{\Gamma_{\tau''}} P_1 \, ds \right) / \int_{\Gamma_K} dx_2 = \text{const}.$$

In fact, using  $T_1(\bar{\lambda}) = T_1(\bar{\lambda}) = 0$  on  $\Gamma_0$  and  $dx_2 = n_1'' \, ds$ ,  $T_1(\bar{\lambda}) = 0$  on  $\Gamma_K$ , we realize that the choice (2.2), (2.3) satisfies the necessary equilibrium conditions (2.1). Moreover, the necessary condition for the existence of a solution is

$$(2.4) \quad V_1 = \int_{\Omega''} F_1 \, d\mathbf{x} + \int_{\Gamma_{\tau''}} P_1 \, ds \geq 0$$

(cf. Remark 1.2 and (1.16)<sub>1</sub>). Hence  $g \leq 0$  and (1.22) is also true.

Example 2. For the same  $\Gamma_0$  and  $\Gamma_K$  with  $n_2'' > 0$  on  $\Gamma_K$ , we choose (2.2) with  $g = 0$  and

$$(2.5) \quad T_i(\bar{\lambda}') = T_i(\bar{\lambda}'') = g_1 = -V_1 \Big/ \int_{\Gamma_K} dx_1 \quad \text{on } \Gamma_K.$$

Obviously,  $\bar{\lambda} \notin K_{F,p}^+$ , unless  $V_1 = 0$ .

Example 3. Let  $\Gamma_0 = \emptyset$  and  $\Gamma_K$  be a straight segment parallel with  $x_1$ -axis. We can choose  $\bar{\lambda}$  such that

$$(2.6) \quad T_n(\bar{\lambda}'') = -T_2(\bar{\lambda}'') = V_2 \Big/ \int_{\Gamma_K} ds = g \quad \text{on } \Gamma_K,$$

where  $V_2 = \int_{\Omega'} F_2 d\mathbf{x} + \int_{\Gamma_K'} P_2 ds$ . Since  $V_2 \leq 0$  is necessary for the existence of a solution, we have  $T_n(\bar{\lambda}'') \leq 0$  on  $\Gamma_K$  and (1.22) can also be satisfied.

Remark 2.1. It is not essential for  $\bar{\lambda}$  to belong to  $K_{F,p}^+$ . From the practical point of view, however,  $\bar{\lambda}$  should be such that  $T_i(\bar{\lambda})$  are constant on  $\Gamma_K$ . Then we can construct internal approximations of  $K_{F,p}^+$ , which is advantageous.

Remark 2.2. An algorithm for finding  $\bar{\lambda}$  will be considered in Section 3.

Henceforth we choose the configuration of Example 1 to show the approximations of  $K_{F,p}^+$  in detail. Let the stress field  $\bar{\lambda}$  satisfy the following condition

$$(2.7) \quad \int_{\Omega} \bar{\lambda}_{ij} e_{ij}(\mathbf{v}) d\mathbf{x} = L(\mathbf{v}) + \int_{\Gamma_K} g(v_n' + v_n'') ds \quad \forall \mathbf{v} \in V.$$

Obviously, it holds

$$\lambda \in K_{F,p}^+ \Leftrightarrow \lambda - \bar{\lambda} \equiv \tau \in \mathcal{U}_0,$$

where

$$\mathcal{U}_0 = \left\{ \tau \in S \mid \int_{\Omega} \tau_{ij} e_{ij}(\mathbf{v}) d\mathbf{x} \geq -g \int_{\Gamma_K} (v_n' + v_n'') ds \quad \forall \mathbf{v} \in K \right\}.$$

**Lemma 2.2.** 1° Let  $\tau \in \mathcal{U}_0$  be sufficiently smooth. Then  $\tau$  satisfies homogeneous equations (1.18), (1.19), (1.20), (1.21) and

$$(2.8) \quad T_n(\tau') = T_n(\tau'') \leq -g \quad \text{on } \Gamma_K.$$

2° Let  $\Omega$  be divided into a finite number of closed subdomains  $K_r$ ,

$$\Omega = \bigcup_r K_r, \quad \hat{K}_r \cap \hat{K}_s = \emptyset \quad \text{if } r \neq s.$$

Let  $\tau \in S$  satisfy the homogeneous equations (1.18) in every  $K_r$ , homogeneous boundary conditions (1.19), (1.20), (1.21) and (2.8). Moreover, let the stress vector  $\mathbf{T}(\tau)$  be continuous across any common boundary of two adjoint subdomains, i.e.

$$\mathbf{T}(\tau)|_{K_r} + \mathbf{T}(\tau)|_{K_s} = \mathbf{0} \quad \text{on } K_r \cap K_s \quad \forall r \neq s.$$

Then  $\tau \in \mathcal{U}_0$ .

Proof is analogous to that of Lemma 1.6.

Remark 2.3. If the condition (2.4) holds,  $\mathcal{U}_0$  is non-empty, since it contains the zero element. Moreover,  $\mathcal{U}_0$  is convex and closed subset of  $\mathcal{S}$ .

Substituting  $\lambda = \bar{\lambda} + \tau$  into the definition (1.17) of the dual problem, we obtain an *equivalent dual problem*: to find  $\tau^0 \in \mathcal{U}_0$  such that

$$(2.9) \quad J(\tau^0) \leq J(\tau) \quad \forall \tau \in \mathcal{U}_0,$$

where

$$J(\tau) = \frac{1}{2} \int_{\Omega} a_{ijkl} \tau_{ij} (\tau_{kl} + 2\bar{\lambda}_{kl}) \, d\mathbf{x}.$$

## 2.1. AN EQUILIBRIUM FINITE ELEMENT MODEL

Since the admissible stress functions in the equivalent dual problem (2.9) have to satisfy the homogeneous equations (1.18) of equilibrium, we need some finite-dimensional subspaces with the same property. To this end, we can employ an equilibrium stress field model proposed by Watwood and Hartz [3].

The model consists of triangular block-elements, each of them being generated by connecting the vertices of a general triangle with its centre of gravity. On each subtriangle 3 linear functions – components of a self-equilibrated stress field are defined. The stress vector has to be continuous when crossing any common boundary of two subtriangles.

For an arbitrary triangle  $K$  let us define the space of self-equilibrated linear stress fields as follows

$$M(K) = \{ \tau_{11} = \beta_1 + \beta_2 x_1 + \beta_3 x_2, \tau_{22} = \beta_4 + \beta_5 x_1 + \beta_6 x_2, \\ \tau_{12} = \tau_{21} = \beta_7 - \beta_6 x_1 - \beta_2 x_2 \},$$

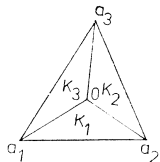


Fig. 1

where  $\beta \in R^7$  is an arbitrary vector. It is readily seen that  $\partial \tau_{ij} / \partial x_j = 0$  in  $K$  for any  $\tau \in M(K)$ ,  $i = 1, 2$ . Next let us consider a triangular block-element  $K = \bigcup_{i=1}^3 K_i$  (see Fig. 1) and define

$$N(K) = \{ \tau = (\tau^1, \tau^2, \tau^3) \mid \tau^i = \tau|_{K_i} \in M(K_i),$$

$$\mathbf{T}(\tau^i) + \mathbf{T}(\tau^{i-1}) = 0 \text{ on } 0a_i, \quad i = 1, 2, 3, \text{ modulo } 3 \}.$$

The last equations mean that the stress vectors are continuous across any side  $0a_i$ .

**Theorem 2.1.** *Let an arbitrary external loading  $\mathbf{T}^0$  of the triangle  $K$  be given,*

such that (i) it varies linearly along the sides of  $K$ , and (ii) it satisfies the global equilibrium conditions.

Then a unique stress field  $\tau \in N(K)$  exists such that

$$\mathbf{T}(\tau) = \mathbf{T}^0 \quad \text{on } \partial K .$$

Moreover the following estimate holds

$$(2.10) \quad \max_{i=1,2,3} \|\tau^i\|_{C(K_i)} \leq c(\alpha) \max_{\substack{s \in \partial K \\ k=1,2}} |T_k^0(s)| ,$$

where the constant  $c(\alpha)$  depends on the minimal angle  $\alpha$  of  $K$  only.

For the proof – see [4] – Theorem 2.1.

Let  $G$  be a bounded polygonal domain,  $h \in (0, 1)$  a parameter,  $\mathcal{T}_h$  a triangulation of  $G$ . Define

$$h = \max_{K \in \mathcal{T}_h} \text{diam } K ,$$

$$N_h(G) = \{ \tau \in S \mid \tau|_K \in N(K) \quad \forall K \in \mathcal{T}_h, \quad \mathbf{T}(\tau)|_K + \mathbf{T}(\tau)|_{K'} = 0 \quad \text{on any } \bar{K} \cap \bar{K}' \} ,$$

$$E(G) = \{ \tau \in [H^1(G)]^4 \mid \tau_{12} = \tau_{21}, \quad \partial \tau_{ij} / \partial x_j = 0 \quad \text{in } G, \quad i = 1, 2 \} .$$

**Theorem 2.2.** *There exists a linear continuous mapping  $r_h : E(G) \rightarrow N_h(G)$  such that for any  $\tau \in E(G) \cap [H^2(G)]^4$*

$$(2.11) \quad \|\tau - r_h \tau\|_{[L_2(G)]^4} \leq C h^2 \|\tau\|_{[H^2(G)]^4}$$

holds, where  $C$  does not depend on  $h$  and  $\tau$ , provided the family  $\{\mathcal{T}_h\}$  of triangulations is regular (cf. [1] – II, Section 2.1).

For the proof – see [4] – Theorem 2.5.

Remark 2.4. The estimate (2.11) follows also from some results of Johnson and Mercier [5].

The mapping  $r_h$  from Theorem 2.2 is defined “locally” on every triangular block-element  $K \in \mathcal{T}_h$  as follows:

The stress vector components  $T_k(\tau)$  on any side  $S_j$  of  $K$  are projected in  $L_2(S_j)$  onto the space  $P_1(S_j)$  of linear functions. By these projections  $\mathbf{T}^0$ , the stress field  $r_h \tau|_K \in N(K)$  is determined uniquely, by virtue of Theorem 2.1.

Remark 2.5. Any stress field  $\tau \in N_h(G)$  satisfies the equilibrium equations  $\partial \tau_{ij} / \partial x_j = 0$ ,  $i = 1, 2$  in the sense of distributions.

## 2.2. APPROXIMATIONS OF THE DUAL PROBLEM BY AN EQUILIBRIUM FINITE ELEMENT MODEL

Assume that both  $\Omega'$  and  $\Omega''$  are bounded polygonal domains. We define an approximation of  $\mathcal{U}_0$  as follows:

$$\mathcal{U}_{0h} = \mathcal{U}_0 \cap N_h(\Omega) ,$$

where

$$N_h(\Omega) = \{(\tau', \tau'') \mid \tau^M \in N_h(\Omega^M), M = ', ''\}.$$

We say that  $\tau^h \in \mathcal{U}_{0h}$  is an approximation of the dual problem, if

$$(2.12) \quad J(\tau^h) \leq J(\tau) \quad \forall \tau \in \mathcal{U}_{0h}.$$

**Lemma 2.3.** *If (2.4) holds, there exists a unique solution of the problem (2.12).*

*Proof.* Obviously,  $N_h(\Omega) \subset S$  is a linear and finite-dimensional subset, therefore it is closed and convex. Using Remark 2.3, we conclude that also  $\mathcal{U}_{0h}$  is closed, convex and non-empty. Since the functional  $J$  is differentiable and strictly convex, the existence and uniqueness of  $\tau^h$  follows easily.

An algorithm for finding  $\tau^h$  will be presented in the next Section. Here we shall try to estimate the error

$$\|\lambda - \lambda^h\|_{0,\Omega} = \|\tau^0 - \tau^h\|_{0,\Omega}$$

where

$$\lambda = \bar{\lambda} + \tau^0, \quad \lambda^h = \bar{\lambda} + \tau^h, \quad \|\cdot\|_{0,\Omega} = \|\cdot\|_{[L^2(\Omega)]^4}.$$

To this end we employ a lemma of Mosco and Strang [6].

**Lemma 2.4.** *Let an element  $W^h \in \mathcal{U}_{0h}$  exist such that*

$$(2.13) \quad 2\tau^0 - W^h \in \mathcal{U}_0.$$

*Then it holds*

$$(2.14) \quad \|\tau^0 - \tau^h\|_{0,\Omega} \leq C\|\tau^0 - W^h\|_{0,\Omega}.$$

For the proof – see [6] or [7] – Lemma 2.1.

Hence it remains to construct a  $W^h \in \mathcal{U}_{0h}$ , satisfying (2.13) and sufficiently close to  $\tau^0$ .

**Theorem 2.3.** *Let  $\Gamma_0$  consist of straight segments parallel with  $x_1$ -axis and  $\Gamma_K$  be a straight segment such that  $n_1'' > 0$  on  $\Gamma_K$ .*

*Assume that  $\tau^0|_{\Omega^M} \in [H^2(\Omega^M)]^2$ ,  $M = \text{I, II}$ , and  $T_n(\tau^0) \in H^2(\Gamma_K)$ . Let the family of triangulations  $\{\mathcal{T}_h\}$ ,  $0 < h \leq 1$ , satisfy the following conditions:*

- (A1) *it is regular (cf. [1] – II, Section 2.1);*
- (A2) *a positive number  $\beta$  exists, independent of  $h$  and such that the ratio of any two sides in  $\mathcal{T}_h$  is less than  $\beta$ ;*
- (A3) *between  $\Gamma_K$  and  $\Gamma_0$  in  $\Omega''$  and between  $\Gamma_K$  and  $\Gamma_u$  in  $\Omega'$  the triangulation  $\mathcal{T}_h$  is inscribed into smooth “vaulted strips” with bounded curvature and bounded slope  $|\vartheta| \leq \Theta < \pi/2$ , ( $\Theta$  independent of  $h$ ), which are perpendicular to  $\Gamma_0$  and  $\Gamma_K$  – see Fig. 2.*

Then

$$\|\tau^0 - \tau^h\|_{0,\Omega} \leq C(\tau^0) h^{3/2},$$

where  $C(\tau^0)$  does not depend on  $h$ .

Proof is based on Lemma 2.4 and the two following lemmas.

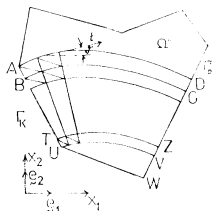


Fig. 2

**Lemma 2.5.** Assume that  $\tau^0 \in H^2(\Omega^M)$ ,  $M = 1, 2$ ,  $T_n(\tau^0) \in H^2(\Gamma_K)$ . Then a piecewise linear function  $\psi^h$  on  $\Gamma_K$  exists with the nodes determined by the triangulation  $\mathcal{T}_h$  (discontinuous, in general) and such that

$$\begin{aligned} \int_{\Gamma_K} \psi^h ds &= \int_{\Gamma_K} T_n(r_h \tau^0) ds, \\ 2 T_n(\tau^0) + g &\leq \psi^h \leq -g \quad \text{on } \Gamma_K, \\ \|T_n(r_h \tau^0) - \psi^h\|_{0,\Gamma_K} &\leq C h^2 \left\| \frac{d^2}{ds^2} T_n(\tau^0) \right\|_{0,\Gamma_K}, \end{aligned}$$

where  $r_h$  is the mapping from Theorem 2.2.

Proof is analogous to that of Lemma 4.2 in [8].

**Lemma 2.6.** Let the triangulations  $\mathcal{T}_h$  satisfy conditions (A1), (A2), (A3). Given a piecewise linear function  $\varphi^h$  on  $\Gamma_K$ , with the nodes determined by  $\mathcal{T}_h$  and such that

$$(2.15) \quad \int_{\Gamma_K} \varphi^h ds = 0.$$

Then there exists a function  $w^h \in N_h(\Omega)$  such that

$$(2.16) \quad \mathbf{T}(w^h) = 0 \quad \text{on } \Gamma_\tau,$$

$$(2.17) \quad T_t(w^h) = 0 \quad \text{on } \Gamma_0,$$

$$(2.18) \quad T_t(w^{h1}) = T_t(w^{h||}) = 0 \quad \text{on } \Gamma_K,$$

$$(2.19) \quad T_n(w^{h1}) = T_n(w^{h||}) = \varphi^h \quad \text{on } \Gamma_K,$$

$$(2.20) \quad \|w^h\|_{0,\Omega} \leq C h^{-1/2} \|\varphi^h\|_{0,\Gamma_K}.$$

Proof. Consider e.g. the domain  $\Omega''$  and the triangulation  $\mathcal{T}_h$ , satisfying (A1)–(A3) – see Fig. 2, where  $\Gamma_K = AU$ ,  $\Gamma_0 \supset DV$ .

1° Let  $ABCD$  be the upper “vaulted strip”. On the edge  $AB$  we have a force

$$\mathbf{P}_1 = \int_A^B \varphi \, ds$$

(henceforth the superscript  $h$  will be omitted), perpendicular to  $AB$ , acting in  $AB$  and a moment  $M_1$ , where

$$(2.21) \quad |M_1| = \frac{1}{3} l_{AB}^2 \frac{|\varphi(A)|^2}{|\varphi(A)| + |\varphi(B)|}$$

holds if  $\varphi(A) < 0$ ,  $\varphi(B) > 0$ ,  $|\varphi(A)| \leq |\varphi(B)|$  or  $M_1 = 0$  if  $\varphi(A)\varphi(B) \geq 0$ .

The stress field response to the loading  $\mathbf{P}_1$  and  $M_1$  will be estimated separately.

2° *The loading by the force  $\mathbf{P}_1$ .* Let the stress vectors on the upper edge  $AE_1F_1 \dots$  of the strip be zero (Fig. 3). On the lower edge  $BEF \dots$ , however, let a piecewise constant loading  $q_j$  acts, parallel to the  $x_2$ -axis, and such that the resultants  $\mathbf{V}_j = \mathbf{P}_1 - \sum_{i=1}^j q_i l_i \mathbf{e}_2$ ,  $j = 1, 2, \dots$ , are parallel to the tangent  $\mathbf{t}$  of the arc  $AD$  at the points  $E_1, F_1, \dots$ , and act inside the sides  $E_1E, F_1F, \dots$ .

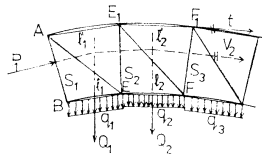


Fig. 3

We can show that *all stress vector components* in the strip  $ABCD$  are bounded above by the number

$$C \max_{\Gamma_K} |\varphi|,$$

where  $C$  is independent of  $h$ .

In fact, we may write (see Fig. 4)

$$(2.22) \quad \begin{aligned} |P_1| &= \frac{1}{2} s_1 |\varphi(A) + \varphi(B)| \leq s_1 \max_{AB} |\varphi|, \\ |V_1| &= |P_1| \cos \vartheta_0 \cos^{-1} \vartheta_1 \leq |P_1| \cdot \cos^{-1} \Theta \\ |Q_1| &= |q_1 l_1| = |P_1| \cos \vartheta_0 (\operatorname{tg} \vartheta_0 - \operatorname{tg} \vartheta_1) = \\ &= |P_1| \cos^{-1} \vartheta_1 \cdot \sin(\vartheta_0 - \vartheta_1) \leq \\ &\leq |P_1| \cos^{-1} \Theta \cdot l_1 \varrho^{-1} \leq C \beta |P_1| \cdot l_1, \end{aligned}$$

where  $\varrho$  denotes the radius of the curvature at the point  $E_1$ .

Consequently, we have

$$(2.23) \quad |q_1| \leq C \beta |P_1| = C_1 h \max_{AB} |\varphi|.$$

The values of  $V_i, q_i, i = 2, 3, \dots$ , can be estimated in a similar way.

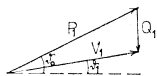


Fig. 4

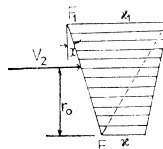


Fig. 5

Next let us consider the stress vectors on the side  $FF_1$ . They are equivalent with the resultant  $V_2$ , which acts at the point  $F_2$ ,  $FF_2 = z_0$ ,  $FF_1 = s_3$  (see Fig. 5). It holds

$$(2.24) \quad \begin{aligned} V_2 &= \frac{1}{2} s_3 (\alpha_1 + \alpha) \\ V_2 r_0 &= \frac{1}{2} s_3 (\alpha_1 r_1 + \alpha r) \end{aligned}$$

where

$$\begin{aligned} r_1 &= \frac{2}{3} s_3 \cos \chi, \quad r = \frac{1}{3} s_3 \cos \chi, \\ r_0 &= z_0 \cos \chi, \quad z_0 < s_3. \end{aligned}$$

The solution of (2.24) with respect to  $\alpha_1, \alpha$  yields that

$$\alpha_1 = \frac{2V_2(r - r_0)}{s_3(r - r_1)}.$$

One easily derives the following inequalities

$$\begin{aligned} \left| \frac{r - r_0}{r - r_1} \right| &= \left| \frac{\frac{1}{3}s_3 - z_0}{\frac{1}{3}s_3} \right| = \left| 1 - 3 \frac{z_0}{s_3} \right| \leq 2, \\ |\alpha_1| &\leq \frac{4}{s_3} |V_2| \leq C \frac{s_1}{s_3} \max_{AB} |\varphi| \leq C \beta \max_{AB} |\varphi|, \end{aligned}$$

where (2.22) has been employed.

A similar estimate can be deduced for  $\alpha$ . The stress vectors on the side  $F_1G$  can be estimated by an analogous way.

3° *The loading by the moment  $M_1$ .* From (2.21) it follows

$$|M_1| \leq \frac{1}{6} s_1^2 \max_{AB} |\varphi|.$$

Let both the upper and lower edge of the strip  $ABCD$  be without loading. Consider any side connecting the upper with the lower edge, e.g.  $FF_1$ . The moment  $M_1$  will be



equivalent with a linear loading, perpendicular to  $FF_1$ , the maximal value of which can be estimated as follows

$$(2.25) \quad |m| = \frac{6}{s_3} |M_1| \leq \left( \frac{s_1}{s_3} \right)^2 \max_{AB} |\varphi| = \beta^2 \max_{AB} |\varphi|.$$

4° Consider an arbitrary “interior vaulted strip”, between the strips  $ABCD$  and  $TUVZ$  – see Fig. 2. We shall construct the stress vectors, bounded again by  $C \max_{\Gamma_K} |\varphi|$ .

Let the upper edge be loaded by a piecewise constant load  $-q_i^H$ , acting “upwards” and the lower edge by a piecewise constant load  $q_i^D$  “downwards”. The differences  $\Delta q_i = q_i^D - q_i^H$  are such that the resultants

$$\mathbf{V}_j^{(n)} = \mathbf{P}_n - \sum_{i=1}^j \Delta q_i l_i \mathbf{e}_2, \quad j = 1, 2, \dots$$

where

$$\mathbf{P}_n = \int_{s_1^{(n)}} \varphi \, ds,$$

are parallel to the corresponding tangent of the upper arc, as previously.

Besides,  $q_i^H$  equals to  $q_i^D$  of the neighbouring strip above, of course.

The resultants  $V_i^{(n)}$  can be estimated as in the strip  $ABCD$ . By an analogy to (2.23), we may write

$$|q_i^D| \leq \sum_{j=1}^n |\Delta q_i^{(j)}| \leq \sum_{j=1}^n Ch \max_{\Gamma_K} |\varphi| \leq C_1 \max_{\Gamma_K} |\varphi|,$$

using the inequalities

$$nh \leq h\beta \min_{j=1,2,\dots,n} s_1^{(n)} \leq \beta \sum_j s_1^{(n)} \leq \beta \text{mes } \Gamma_K.$$

5° The loading by the moment  $M_n$ , acting on the interior vaulted strip, can be estimated as that by  $M_1$  in the upper strip.

6° Finally, let us show that the loading  $q_i^D$  of the lower strip  $TUVZ$  equals to zero if the resultants  $\mathbf{V}_i^{(N)}$  are parallel to the same tangent  $\mathbf{t}$  at the corresponding vertex of the edge  $AE_1 \dots D$ .

Let  $AU = \Gamma_K$  consist of  $N$  sides  $S_1^{(1)}, \dots, S_1^{(N)}$ . The following equations hold for the quadrangles adjoint to  $\Gamma_K$ :

$$\begin{aligned} \mathbf{P}_1 - Q_1^{(1)} \mathbf{e}_2 &= \mathbf{V}_1^{(1)}, \\ \mathbf{P}_2 - \Delta Q_1^{(2)} \mathbf{e}_2 &= \mathbf{V}_1^{(2)}, \\ &\vdots \\ \mathbf{P}_N - \Delta Q_1^{(N)} \mathbf{e}_2 &= \mathbf{V}_1^{(N)}, \end{aligned}$$

where  $\Delta Q_1^{(j)} = Q_1^{(j)} - Q_1^{(j-1)}$ ,  $Q_1^{(j)} = l_1^{(j)} q_1^{D(j)}$ ,  $j = 1, 2, \dots, N$ .

Since (2.15) implies that

$$\sum_{j=1}^N \mathbf{P}_j = \int_{\Gamma_K} \varphi \, ds = 0,$$

by adding we obtain

$$-Q_1^{(N)} \mathbf{e}_2 = \sum_{j=1}^N \mathbf{V}_1^{(j)}.$$

The vectors on both sides have different directions and consequently  $\mathbf{Q}_1^{(N)} = 0$ ,  $\sum_{j=1}^N \mathbf{V}_1^{(j)} = 0$ .

Considering the second column of quadrangles, we prove that  $\mathbf{Q}_2^{(N)} = 0$ , changing only  $\mathbf{P}_j$  for  $\mathbf{V}_1^{(j)}$ ,  $j = 1, 2, \dots, N$ , a.s.o.

7° The triangulation of  $\Omega'$  outside the vaulted strip  $AUVD$  can be chosen arbitrarily, it must fit in the division of the arcs  $\widehat{AD}$  and  $\widehat{UV}$ , of course. We choose zero stress field everywhere outside  $AUVD$ .

8° On the domain  $\Omega'$  a parallel approach can be used.

9° The above method of construction of the stress vectors guarantees that each triangular element  $K$  is loaded by self-equilibrated linearly distributed external forces. Theorem 2.1 can be applied to obtain a uniquely determined stress field  $w^h|_K \in N(K)$  and the estimate (2.10) holds. Since the condition of continuity of stress vector is also satisfied, we arrive at a stress field  $w^h \in N_h(\Omega)$ . From (2.10) and the above estimates it follows

$$\max_{i=1,2,3} \|w^{hi}\|_{C(\bar{K}_i)} \leq c \max_{\Gamma_K} |\varphi^h| \quad \forall K \in \mathcal{T}_h,$$

and the same bound is true for  $\|w^h\|_{0,\Omega}$ .

It holds

$$(2.26) \quad \max_{\Gamma_K} |\varphi^h| \leq C h^{-1/2} \|\varphi^h\|_{L_2(\Gamma_K)}.$$

In fact, let  $AB$  be an arbitrary side of  $\mathcal{T}_h$  on  $\Gamma_K$ . Then

$$\int_A^B \varphi^2 \, ds = \frac{1}{6} l_{AB} [\varphi(A)^2 + \varphi(B)^2 + (\varphi(A) + \varphi(B))^2] \geq \frac{1}{6} l_{AB} [\varphi(A)^2 + \varphi(B)^2],$$

consequently

$$|\varphi(A)|^2 + |\varphi(B)|^2 \leq 6 l_{AB}^{-1} \|\varphi\|_{L_2(AB)}^2 \leq 6\beta h^{-1} \|\varphi\|_{L_2(\Gamma_K)}^2$$

and (2.26) follows.

Thus we obtain the inequality (2.20). By the construction of  $w^h$  (especially of the stress vector  $\mathbf{T}(w^h)$ ) it is easy to verify the boundary conditions (2.16)–(2.19).

Q.E.D.

**Proof of Theorem 2.3.** Let  $\psi^h$  be the function from Lemma 2.5. If we set

$$\varphi^h = T_n(r_h \tau^0) - \psi^h,$$

then Lemma 2.6 implies that a  $w^h \in N_h(\Omega)$  exists, satisfying the boundary conditions (2.16)–(2.19) and the estimate (2.20). Defining  $W^h = r_h \tau^0 - w^h$ , we obtain  $W^h \in \mathcal{U}_{0h}$ . In fact,  $W^h \in N_h(\Omega)$  and, by virtue of the definition of  $r_h$  (cf. Remark 2.4)

$$\begin{aligned} \mathbf{T}(W^h) &= 0 \quad \text{on } \Gamma_\tau, \\ T_i(W^h) &= 0 \quad \text{on } \Gamma_0, \\ T_i(W^h) &= T_i(W^{h||}) = 0 \quad \text{on } \Gamma_K, \\ T_n(W^h) &= T_n(W^{h||}) = T_n(r_h \tau^0) - \varphi^h = \psi^h \leq -g \quad \text{on } \Gamma_K. \end{aligned}$$

Hence (see also Remark 2.5)  $W^h \in \mathcal{U}_0$  follows.

Moreover, we show that

$$(2.27) \quad 2\tau^0 - W^h \in \mathcal{U}_0.$$

In fact, both  $\tau^0$  and  $W^h$  satisfy homogeneous boundary conditions on  $\Gamma_\tau$  for  $T$  and on  $\Gamma_0, \Gamma_K$  for  $T_i$ , respectively. Moreover,

$$T_n(2\tau^0 - W^h) = 2T_n(\tau^0) - \psi^h \leq -g \quad \text{on } \Gamma_K.$$

Consequently, (2.27) follows.

Using Theorem 2.2, we write

$$\begin{aligned} \|\tau^0 - W^h\|_{0,\Omega} &\leq \|\tau^0 - r_h \tau^0\|_{0,\Omega} + \|r_h \tau^0 - W^h\|_{0,\Omega} \leq \\ &\leq Ch^2 \|\tau^0\|_{2,\Omega} + \|W^h\|_{0,\Omega}, \end{aligned}$$

and for the last term Lemmas 2.6 and 2.5 imply

$$\|W^h\|_{0,\Omega} \leq C h^{-1/2} \|\varphi^h\|_{0,\Gamma_K} \leq C_1 h^{3/2} \left\| \frac{d^2}{ds^2} T_n(\tau^0) \right\|_{0,\Gamma_K}.$$

Finally, from Lemma 2.4 the error estimate follows. Q.E.D.

Remark 2.4. Let us discuss also the configuration of Example 3 in short, i.e., let  $\Gamma_0 = \emptyset$  and  $\Gamma_K$  be a straight segment, parallel with the  $x_1$ -axis.

We construct a  $\bar{\lambda}$  satisfying (2.7), define  $\mathcal{U}_0$ , the equivalent dual problem (2.9),  $\mathcal{U}_{0h}$  and the approximations (2.12). If  $V_2 \leq 0$ ,  $\mathcal{U}_0$  contains the zero element and there exists a unique approximation  $\tau^h$ .

We can prove an analogue of Theorem 2.3, where the same regularity of  $\tau^0$  is required, assumptions (A1), (A2) are preserved and (A3) is replaced by the following

(A3'): in  $\Omega''$  the triangulations  $\mathcal{T}_h$  contain a fixed rectangle  $AUBC$ , independent of  $h$ , with  $AU = \Gamma_K$ , which is divided into small rectangular elements. In  $\Omega'$  the triangulations  $\mathcal{T}_h$  satisfy the same conditions as in (A3).

The proof is based on a modified Lemma 2.5, where a moment equilibrium condition

$$\int_{\Gamma_K} \psi^h s \, ds = \int_{\Gamma_K} T_n(r_h \tau^0) s \, ds$$

is added, and on an analogue of Lemma 2.6. To prove the latter lemma, we employ the idea of stress fields in a “beam”  $AUBC$ . Thus we obtain upper bounds  $C \max_{\Gamma_K} |\varphi|$  for the linear stress vectors components on every side, as previously.

### 3. ALGORITHM FOR APPROXIMATIONS OF THE DUAL PROBLEM

In the theoretical analysis we used the stress vector components to determine the stress fields  $\tau \in N_h(\Omega)$  – cf. Theorem 2.1. For practice, however, the stress tensor parameters  $\beta_1^i, \dots, \beta_7^i$  ( $i = 1, 2, 3$ ) (cf. Section 2.1) are more suitable. First we present a survey of some results of Watwood and Hartz [3], which can be employed immediately to construct an algorithm for the solution of the problem (2.12).

On every subtriangle  $K_i$  the stress field is given as follows

$$(3.1) \quad \tau = \begin{bmatrix} \tau_{11} \\ \tau_{22} \\ \tau_{12} \end{bmatrix} = \frac{E}{\sqrt{AE_0}} \begin{bmatrix} \sqrt{A} & 0 & 0 & 0 & x_1 & 0 & x_2 \\ 0 & \sqrt{A} & 0 & x_1 & 0 & x_2 & 0 \\ 0 & \sqrt{0} & \sqrt{A} & 0 & -x_2 & -x_1 & 0 \end{bmatrix} S = MS,$$

where  $S \in \mathbb{R}^7$  is a vector of coefficients,  $E$  Young’s modulus,  $E_0$  a dimensionless quantity, being equal to the reference modulus,  $A$  the area of  $K_i$ .

Let the origin of coordinates  $(x_1, x_2)$  be at the center of gravity of  $K_i$ , the material be homogeneous and isotropic in  $K_i$ , with a Poisson’s constant  $\sigma$ . Then we have

$$(3.2) \quad \int_{K_i} a_{mjk} \tau_{mj} \tau_{kl} \, d\mathbf{x} = S^T f S,$$

where ( $t$  is the thickness of the element  $K_i$ ):

$$f = \frac{tE}{E_0} \begin{bmatrix} \alpha A, \\ \beta A, \alpha A, \\ 0, 0, \gamma A, & \text{symmetry} \\ 0, 0, 0, \alpha \delta_1, \\ 0, 0, 0, \beta \delta_1, \alpha \delta_1 + \gamma \delta_2, \\ 0, 0, 0, \alpha \delta_{12}, (\beta + \gamma) \delta_{12}, \alpha \delta_2 + \gamma \delta_1, \\ 0, 0, 0, \beta \delta_{12}, \alpha \delta_{12}, & \beta \delta_2, & \alpha \delta_2 \end{bmatrix}$$

$$\delta_j = \frac{1}{A} \int_{K_i} (x_j)^2 \, dx_1 \, dx_2, \quad j = 1, 2,$$

$$\delta_{12} = \frac{1}{A} \int_{K_i} x_1 x_2 \, dx_1 \, dx_2;$$

$\alpha, \beta, \gamma$  are constants, specified for plane stress as

$$\alpha = 1, \quad \beta = -\sigma, \quad \gamma = 2(1 + \sigma)$$

and for plane strain as

$$\alpha = 1 - \sigma^2, \quad \beta = -\sigma(1 + \sigma), \quad \gamma = 2(1 + \sigma).$$

Likewise we could derive

$$(3.3) \quad \int_{K_i} a_{mjk_l} \tau_{mj} \bar{\lambda}_{kl} dx = \left( \int_{K_i} \bar{\lambda} B^{-1} M dx \right) S,$$

where  $\bar{\lambda} = (\bar{\lambda}_{11}, \bar{\lambda}_{22}, \bar{\lambda}_{12})^T$ ,  $M$  is the matrix from (3.1) and  $B^{-1}$  is a  $(3 \times 3)$  matrix of the corresponding inverse Hooke's law.

Each of the stress vector components on a side  $a_i a_{i+1}$  can be expressed in terms of external parameters  $S^* \in \mathbb{R}^4$  and a parameter  $p \in \langle -1, 1 \rangle$ , as follows:

$$T_1(p) = S_1^* + S_2^* p,$$

$$T_2(p) = S_3^* + S_4^* p.$$

For example, consider the side  $a_2 a_3$ . Then  $p = -1$  at  $a_2$ ,  $p = 1$  at  $a_3$ ,

$$S^* = \frac{1}{2l_a \sqrt{A}} \stackrel{(a)}{C} S,$$

where  $l_a$  is the length of  $a_2 a_3$  and  $C$  the following  $(4 \times 7)$  matrix:

$$\stackrel{(a)}{C} = \begin{bmatrix} -2\sqrt{(A)}(Y_2 - Y_3), & 0, & 2\sqrt{(A)}(X_2 - X_3) & 0, \\ 0, & 0, & 0, & 0, \\ 0, & 2\sqrt{(A)}(X_2 - X_3), & -2\sqrt{(A)}(Y_2 - Y_3), & X_2^2 - X_3^2, \\ 0, & 0, & 0, & -(X_2 - X_3)^2, \\ -2(X_2 Y_2 - X_3 Y_3), & -(X_2^2 - X_3^2), & -(Y_2^2 - Y_3^2) \\ -2(X_2 - X_3)(Y_2 - Y_3), & (X_2 - X_3)^2, & (Y_2 - Y_3)^2 \\ Y_2^2 - Y_3^2, & 2(X_2 Y_2 - X_3 Y_3), & 0 \\ -(Y_2 - Y_3)^2, & -2(X_2 - X_3)(Y_2 - Y_3), & 0 \end{bmatrix}$$

By a cyclic permutation of indices, the matrices  $C$  and  $C$  can be obtained. Denote the external parameters on the side  $b$   $S_5^*, S_6^*, S_7^*, S_8^*$  and on  $c$   $S_9^*, S_{10}^*, S_{11}^*, S_{12}^*$ . For the total vector  $S^* \in \mathbb{R}^{12}$  it holds

$$(3.4) \quad S^* = CS,$$

where the  $(12 \times 7)$  matrix  $C$  is composed of  $\stackrel{(a)}{C}$ ,  $\stackrel{(b)}{C}$ ,  $\stackrel{(c)}{C}$ :

$$C = \frac{1}{2\sqrt{A}} \begin{bmatrix} l_a^{-1} \stackrel{(a)}{C} \\ l_b^{-1} \stackrel{(b)}{C} \\ l_c^{-1} \stackrel{(c)}{C} \end{bmatrix}.$$

The conditions of continuity for the stress vector across any common side of adjoint triangles take the form

$$(3.5) \quad S_i^* + S_j^* = 0,$$

(where the indices  $i, j$  correspond with the same basis functions but with different triangles).

It is readily seen from the definition of  $\mathcal{U}_{0h} = \mathcal{U}_0 \cap N_h(\Omega)$ , that  $\tau \in \mathcal{U}_{0h}$  if and only if:

all constraints of the type (3.5) hold,

$$(3.6) \quad S_j^* = 0 \quad \text{on any side } a_i a_{i+1} \subset \bar{\Gamma}_\tau,$$

$$(3.7) \quad S_j^* t_1 + S_{j+2}^* t_2 = 0 \quad \text{on } \bar{\Gamma}_0 \cup \bar{\Gamma}_K,$$

(where  $t_k$  denote the tangential vector components and (3.7) hold independently for  $\tau^M \in N_h(\Omega^M)$ ,  $M = 1, \dots, \parallel$ ),

$$(3.8) \quad \begin{aligned} [S_j^* n'_1 + S_{j+2}^* n'_2 - (S_{j+1}^* n'_1 + S_{j+3}^* n'_2)]_{\Omega'} &\leq -g \quad \text{on } \bar{\Gamma}_K, \\ [S_j^* n'_1 + S_{j+2}^* n'_2 + (S_{j+1}^* n'_1 + S_{j+3}^* n'_2)]_{\Omega'} &\leq -g \quad \text{on } \bar{\Gamma}_K \end{aligned}$$

and conditions (3.5) for any common side of two triangles, which belongs to  $\bar{\Gamma}_K$ .

The following *compensation of parameters* is recommended. In each triangular block-element we exclude the internal degrees of freedom. Let the conditions of continuity at the segment  $0a_i$  (see Fig. 1) be written as

$$(3.9) \quad A_u S = 0$$

where  $A_u$  is a  $(12 \times 21)$  matrix and  $S$  a  $(21 \times 1)$  matrix. There exists a regular  $(21 \times 21)$  matrix  $Q$  such that

$$(3.10) \quad A_u Q = [I \mid 0],$$

where  $I$  is the unit matrix. The matrix  $Q$  is not determined uniquely. It suffices even to replace  $I$  in (3.10) by any regular  $(12 \times 12)$  matrix. Moreover, only the last nine columns of  $Q$ , i.e. a matrix  $Q_1$ , is needed.

Let us transform  $S$  as follows

$$(3.11) \quad S = Q \hat{S} = [Q_0 \mid Q_1] \begin{bmatrix} \hat{S}_u \\ \hat{S}_i \end{bmatrix},$$

where  $Q$  is divided between 12th and 13th column and  $\hat{S}$  accordingly. Substituting into (3.9), we obtain

$$A_u Q \hat{S} = [I \mid 0] \hat{S} = 0 \Rightarrow \hat{S}_u = 0$$

and the transformation (3.11) is reduced to

$$(3.12) \quad S = Q_1 \hat{S}_i,$$

where  $Q_1$  is a  $(21 \times 9)$  matrix and  $\hat{S}_l$  a  $(9 \times 1)$  matrix. The parameters  $\hat{S}_l$  are independent degrees of freedom of the block-element.

The functional  $J(\tau)$  of the equivalent dual problem can be evaluated in terms of  $\hat{S}_l$ :

$$\begin{aligned} J(\tau) &= \frac{1}{2} \sum_{K \in \mathcal{T}_h} \sum_{i=1}^3 \int_{K_i} a_{mjkl} \tau_{mj} (\tau_{kl} + 2\bar{\lambda}_{kl}) \, d\mathbf{x} = \\ &= \sum_{K \in \mathcal{T}_h} \sum_{i=1}^3 (\frac{1}{2} S^T f S + B_0^T S) = \sum_{K \in \mathcal{T}_h} (\frac{1}{2} S^T F S + b_0^T S) = \\ &= \sum_{K \in \mathcal{T}_h} (\frac{1}{2} \hat{S}_l^T Q_1^T F Q_1 \hat{S}_l + b_0^T Q_1 \hat{S}_l) = \frac{1}{2} \hat{S}_l^T A \hat{S}_l + b^T \hat{S}_l = J(\hat{S}_l), \end{aligned}$$

(where the vectors  $S, \hat{S}_l$  correspond with a subtriangle, block-element and the whole triangulation, respectively). The  $(N \times N)$  matrix  $A$  is positive definite.

Inserting (3.4) and (3.12) into the conditions of the type (3.5)–(3.8), we obtain the constraints

$$(3.13) \quad D \hat{S}_l = 0,$$

$$(3.14) \quad E \hat{S}_l \leq -\mathbf{g},$$

where  $D$  and  $E$  are  $(p_1 \times N)$  and  $(p_2 \times N)$  matrices, respectively,  $\mathbf{g}$  a vector with identical components, equal to  $g$ .

Let us define the set

$$\mathcal{B} = \{ \hat{S}_l \in \mathbb{R}^N \mid (3.13) \text{ and } (3.14) \text{ are satisfied} \}.$$

Thus we arrived at the problem to find  $\sigma \in \mathcal{B}$  such that

$$(3.15) \quad J(\sigma) \leq J(\hat{S}_l) \quad \forall \hat{S}_l \in \mathcal{B}.$$

One can apply e.g. the *algorithm of Uzawa* for solving (3.15) (cf. [2] – chpt. 5). Denote  $p = p_1 + p_2$  and

$$B = \begin{bmatrix} D \\ E \end{bmatrix}, \quad G = \begin{bmatrix} 0 \\ \mathbf{g} \end{bmatrix}$$

the  $(p \times N)$  and  $(p \times 1)$  matrix, respectively. Introduce the set of admissible Lagrange multipliers

$$A = \{ z \in \mathbb{R}^p \mid z_j \geq 0 \text{ for } j = p_1 + 1, \dots, p \}.$$

Choosing a  $z^0 \in A$ , we calculate  $s^0 \in \mathbb{R}^N$  from the system

$$A s^0 = -b - B^T z^0.$$

Having  $z^n, s^n$ , the values of  $z^{n+1}, s^{n+1}$  are determined as follows

$$\begin{aligned} z^{n+1} &= P_A [z^n + \varrho(B s^n + G)], \\ A s^{n+1} &= -b - B^T z^{n+1}, \end{aligned}$$

where  $P_A$  denotes the projection onto the set  $A$ , i.e.,

$$y = P_A v \Leftrightarrow \begin{cases} y_j = v_j, & j = 1, \dots, p_1 \\ y_j = \max\{0, v_j\}, & j = p_1 + 1, \dots, p, \end{cases}$$

and  $\varrho$  is a sufficiently small parameter.

It can be proven that  $s^n \rightarrow \sigma$  in  $\mathbb{R}^N$  for  $n \rightarrow \infty$ , where  $\sigma$  is the (unique – see Lemma 2.3) solution of (3.15), provided that  $\text{rang } B = p_1 + p_2$ .

Finally, let us discuss again a possible construction of the auxiliary stress field  $\bar{\lambda}$  (cf. Section 2). Again, we consider the configuration of Example 1 and choose the boundary condition (2.2) on  $\Gamma_K$ . According to the interpretation of the set  $K_{F,P}^+$  and (2.7) (see also Lemma 1.6 and Lemma 2.2), we may proceed as follows.

Choose  $\lambda^{(1)} \in S$ , satisfying the equations (1.18) in  $\Omega = \Omega' \cup \Omega''$  (by simple integration with respect to  $x_1$  or  $x_2$ ).

Assume that  $\mathbf{F}$  is a constant vector field (zero or gravitational forces in most cases) and  $\mathbf{P}$  is a piecewise linear vector field. Then  $\lambda_{ij}^{(1)}$  are linear polynomials over  $\Omega$  and  $T_i(\lambda^{(1)})$  are linear on every side of the polygonal boundaries  $\partial\Omega' \cup \partial\Omega''$ .

Setting  $\bar{\lambda} = \lambda^{(1)} + \lambda^{(2)}$  we may look for  $\lambda^{(2)}$  in the space  $N_n(\Omega)$ , satisfying moreover the following boundary conditions

$$(3.16) \quad \mathbf{T}(\lambda^{(2)}) = \mathbf{P} - \mathbf{T}(\lambda^{(1)}) \quad \text{on } \Gamma_\tau,$$

$$(3.17) \quad T_i(\lambda^{(2)}) = -T_i(\lambda^{(1)}) \quad \text{on } \Gamma_0,$$

$$(3.18) \quad T_i(\lambda^{(2)\parallel}) = T_i(\lambda^{(2)\parallel}) = -T_i(\lambda^{(1)}) \quad \text{on } \Gamma_K,$$

$$(3.19) \quad T_n(\lambda^{(2)\parallel}) = T_n(\lambda^{(2)\parallel}) = g - T_n(\lambda^{(1)}) \quad \text{on } \Gamma_K.$$

Since the right-hand sides in (3.16)–(3.19) are linear or piecewise linear functions, the stress field  $\lambda^{(2)}$  exists. It can be constructed using the procedure described above. We employ the parameters  $\hat{S}_i$ , and the formulas (3.1), (3.12), (3.4), write the continuity conditions of the type (3.5) and express the boundary conditions (3.16)–(3.19) in terms of  $\hat{S}_i$  via (3.4) and (3.12). The undetermined values of  $T_n(\lambda^{(2)})$  on  $\Gamma_0$  and  $T_i(\lambda^{(2)})$  on  $\Gamma_u$ ,  $i = 1, 2$ , can be chosen in such a way that the resulting system of linear equations is solvable. The choice corresponds with the global equilibrium of the body  $\Omega''$  and  $\Omega'$ , respectively.

**Remark 3.1.** The dual analysis enables us to find a *posteriori* error estimates and two-sided bounds of the exact energy. The derivation may follow e.g. the approach of [8] – Section 5.

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## Souhrn

### KONTAKT PRUŽNÝCH TĚLES — III. DUÁLNÍ ANALÝZA METODOU KONEČNÝCH PRVKŮ

JAROSLAV HASLINGER, IVAN HLAVÁČEK

Jednostranná kontaktní úloha dvou pružných těles s omezeným rozsahem kontaktu je formulována v napětích pomocí principu minima doplňkové energie. Definují se aproximace řešení, které se skládají z trojúhelníkových blokových rovnovážných prvků. V případě, že přesné řešení je dostatečně regulární, odvozuje se odhad chyby v normě  $L^2$ .

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